

## LAMPIRAN

### REAKTOR

Jenis	= Reaktor alir tangki Berpengaduk (CSTR)
Fase	= Cair - Gas
Bentuk	= Tangki Silinder
Jumlah	= 1
Head	= Torispherical dished head 342, Brownell & Young
Bahan	= Stainless Steel SA 167 grade 10 tipe 310 (Apendix D, item 4, halaman342, Brownell & Young)
Suhu Operasi	= 35 °C
Tekanan	= 3 atm
Waktu Tinggal ( $\theta$ )	= 35 hari = 840 jam
Laju Alir Massa	= 37.224,6000 Kg/jam
Densitas Kotoran Sapi	= 1200 Kg/m <sup>3</sup>
Volume Cairan	= 226057,2200 m <sup>3</sup>

#### A. Menghitung Dimensi Reaktor

Perancangan reaktor dibuat dengan over design sebesar 20%, sehingga volume reaktor menjadi :

$$\text{Volume reaktor} = 1,2 \times \text{volume cairan}$$

$$\text{Volume reaktor} = 1,2 \times 22657,2200 \text{ m}^3$$

$$\text{Volume reaktor} = 31.268,664 \text{ m}^3$$

1. Menghitung diameter dan tinggi reaktor

Reaktor yang digunakan berbentuk silinder tegak

$$\begin{aligned}\text{Volume} &= \text{volume silinder} + \text{volume tutup} \\ &= \text{volume silinder} + 2 \text{ volume } \textit{head}\end{aligned}$$

Tutup berbentuk *torispherical dished head*

Dengan :



$$\text{Volume head} = 0,000049 d^3$$

Sehingga :

$$\text{Volume} = \left( \frac{1}{4} \times \pi \times D^2 \times H \right) + [2 \times (0,000049) \times (D^3)]$$

$$\text{Diameter tangki} = \left( \frac{31268,664}{5,420598} \right)^{1/3}$$

$$\text{Diameter tangki} = 18,3943202 \text{ m} = 60,3488 \text{ ft}$$

$$\text{Perbandingan diameter dan tinggi reactor} = 2.5 : 1$$

Maka tinggi reaktor :

$$H = 7,357728063 \text{ m} = 24,1395 \text{ ft}$$

Diambil ukuran standar tangki : ( App., E, Brownell)

$$\text{Diameter standar} = 70 \text{ ft} = 840 \text{ in} = 21,3360 \text{ m}$$

$$\text{Tinggi standar} = 30 \text{ ft} = 360 \text{ in} = 9,1440 \text{ m}$$

$$\text{Sehingga kapasitas standar tangki sebesar} = 3268,77272 \text{ m}^3$$

### **Menentukan Tebal *shell***

$$t_s = \frac{P \cdot r_i}{f \cdot E - 0,6P} + C \quad \text{Eq. 14.34 (Brownell \& Young. 275)}$$

ts:tebal shell, in

ri:jari-jari shell, (D/2), in

f:allowable stress, psi (Tabel 13.1 Brownell 251)

E:joint efisiensi tipe double-butt weld (0.8) (Tabel 13.2 Brownell 254)

C:faktor koreksi, in (Tabel 6. Timmerhaus, 1991: 542)

P:internal pressure, lb/in<sup>2</sup>

Diketahui:

Allowble working stres (f) = 18750 psi

Joint efficieny (E) = 0.8

Faktor korisi (C)=0.125

umur alat (n)=10 tahun

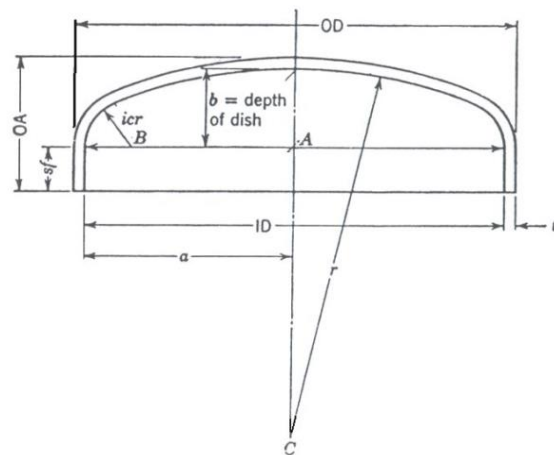
Sehingga, Tebal Shell yang diperoleh adalah:

$t_s=1,1135$  in

$t_s$  standar=  $1(1/8)$ in= $1,125$  in

## B. Menentukan Head Design

### 1. Menentukan jenis head



Head yang digunakan adalah jenis *Torispherical dished head*

Keterangan :

T = Tebal head, in

icr = Inside corner radius, in

r = Radius of dish, in

OD = Outside diameter, in

ID = Inside diameter, in

b = Depth of dish, in

sf = Straight flange

2. Material : Stainless Steel SA 167 grade 10 tipe 310

3. Menentukan tekanan desain reaktor

$$P_{\text{operasi}} = 3 \text{ atm} = 44,1 \text{ psia}$$

$$P_{\text{hidrostatik}} = \frac{\rho \left( \frac{g}{gc} \right) H_L}{144} = 12,5582 \text{ psi}$$

Tekanan desain 5 – 10% di atas tekanan kerja normal/absolut (Coulson, 1983). Tekanan desain diambil 5% di atas tekanan kerja normal.

$$P_{\text{desain}} = 1,05 \times (P_{\text{operasi}} + P_{\text{hidrostatik}})$$

$$P_{\text{desain}} = 1,05 \times (44,1 + 12,5582)$$

$$P_{\text{desain}} = 59,43 \text{ psi}$$

4. Menentukan tebal head (th)

$$t_h = \frac{PD}{(2fE - 0,2P)} + c$$

$$f = 18750 \text{ psi} \quad (\text{Brownell \& Young, 1959})$$

$$E = 0.8 \quad (\text{double welded butt joint})$$

$$c = 0.125 \text{ in}$$

$$D = 1680 \text{ in} \quad (\text{Brownell \& Young, 1959})$$

$$P_{\text{operasi}} = 3 \text{ atm}$$

$$\text{diperoleh Tebal Head (th)} = 1.113 \text{ in}$$

$$\text{th standar} = 1(1/8) = 1.125 \text{ in}$$

5. Menentukan tinggi head ( $h_h$ )

$$t_h = \frac{0,885 \cdot P \cdot r_c}{f \cdot E - 0,1 P}$$

P = Tekanan Perancangan, Psi

f = Tekanan maksimum yang diijinkan pada bahan, Psi

E = Welded joint efficiency

rc = radius of crown

th = tebal head

icr = inside radius corner

rc = OD (Brownell & Young hal 88)

icr = 6% x OD (For torispherical dished head in which the knuckle radius is 6% of the inside crown radius)

$$OD = ID + 2t$$

$$= 1682.25 \quad \text{in}$$

$$rc = 1682.25 \quad \text{in}$$

$$icr = 100.935 \quad \text{in}$$

$$\text{jika, } OD_s = 1680 \quad \text{in}$$

$$th = 1.125 \quad \text{in}$$

dari tabel 5.7 Brownell hal.90 didapat:

$$icr = 100.94 \quad \text{in}$$

$$r = 1682.25 \quad \text{in}$$

$$\begin{aligned}
 a &= ID_s/2 = 840 \text{ in} \\
 AB &= a - icr = 739.065 \text{ in} \\
 BC &= r - icr = 1581 \text{ in} \\
 AC &= (BC^2 - AB^2)^{1/2} = 1397.977 \text{ in} \\
 b &= r - AC = 284.273 \text{ in}
 \end{aligned}$$

Dari tabel 5.6 Brownell hal.88 dengan  $th = 5/8 \text{ in}$  didapat  $sf = 1.5 - 3.5 \text{ in}$

perancangan digunakan  $sf = 3 \text{ in}$

Sehingga, tinggi head ( $H_h$ ) yang diperoleh adalah:

$$\begin{aligned}
 H_h &= th + b + sf \\
 &= 288.398 \text{ in} \\
 &= 24.033 \text{ ft} \\
 &= 7.325 \text{ m}
 \end{aligned}$$

Tinggi total tangki penyimpanan U.K :

$$\text{tinggi tangki + tutup} = 14.641 \text{ m}$$

### C. Menghitung Dimensi Pengaduk

Volume cairan yang diaduk =  $26057.2200 \text{ m}^3$

Jenis pengaduk yang dipilih yaitu *Flat Six-Blade Turbine Impeller with disk*

Perancangan untuk pengadukan dilakukan dengan prinsip similaritas menggunakan model sesuai dengan referensi buku Brown

$$\frac{Da}{Dt} = 1:3 \quad \frac{L}{Da} = 1:4 \quad \frac{J}{Dt} = 1:12 \quad \frac{E}{Da} = 1:1 \quad \frac{W}{Da} = 1:5$$

Dimana:



Dt = Diameter tangki      W = Lebar blade

Da = Diameter impeller      L = Panjang blade

E = Tinggi impeller      J = Lebar baffle

Maka diperoleh :

- a. Diameter Impeller (Da) =  $\frac{1}{3} \times 21,3360 \text{ m} = 7,112 \text{ m}$
- b. Tinggi pengaduk dari dasar tangki (E) = Da = 7,112 m
- c. Lebar baffle (J) =  $1/2 \text{ Dt} = 1,778 \text{ m}$
- d. Lebar blade (W) =  $1/5 \times \text{Da} = 1,422 \text{ m}$
- e. Panjang blade (L) =  $1/4 \text{ Da} = 1,778 \text{ m}$
- f. Kecepatan pengadukan, = 0,1 Putaran/detik (Tabel 10.2 Wallas)

Menghitung nilai Re :

$$\text{Re} = \frac{\rho \times N \times \text{Di}^2}{\mu}$$

$$\text{Re} = \frac{1200 \times 0,1 \times (7,112)^2}{0,97}$$

$$\text{Re} = 6257,386887$$

Dari grafik 10.6 hal 292 (Wallas) diperoleh NP = 3

Sehingga :

$$P = \frac{N^3 \times \text{Di}^5 \times \rho \times \text{Po}}{gc}$$

$$P = 65503.007 \text{ watt}$$

$$P = 87,84 \text{ Hp}$$

Effisiensi motor  $\eta = 80 \%$  (Peters,dkk 2004)

$$\text{Daya motor} = 87,84 \times 0,8 = 109,801 \text{ Hp}$$

Sehingga, diambil daya motor standar 125 Hp